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Saving with Variable Speed Kitchen Exhaust Fans and Make- Up Fans

Kitchen exhaust hoods having fans outfitted with variable speed motors are now offered by commercial kitchen equipment suppliers. Manufacturer's have claimed a payback period of approximately 1-2 years, maybe less. Of course, it all really depends upon the particular details for each project.



A good idea?

Well, there are many aspects to consider, including the following:

- Energy Savings Available
- Overall Safety/Compliance with Codes
- Installation Considerations
- Any Maintenance Concerns

In this article we explore the first of these – Energy Savings. In subsequent articles we will attempt to explore the others.

**“ \$88.15 per month.
Now that's a nice savings.”**

Energy Savings \$

First, there is a considerable savings realized from slowing down fan motors.

Fan Laws say it all. Consider the following:

1. Flow Rate (Q) is proportional to Fan Speed (RPM)

$$\frac{Q_1}{Q_2} = \frac{RPM_1}{RPM_2}$$

2. Fan Pressure induced is proportional to (Fan Speed)²

$$\frac{P_1}{P_2} = \left[\frac{RPM_1}{RPM_2} \right]^2$$

3. Fan Power required is proportional to (Fan Speed)³

$$\frac{BHP_1}{BHP_2} = \left[\frac{RPM_1}{RPM_2} \right]^3$$

For example, a fan that is slowed to ½ of its original speed, will require only (1/2)³ = 1/8 of the original brake horsepower. Hence, a savings of 7/8 or 87.5% of the original power.

A savings in brake horsepower at the exhaust fan is a savings on the monthly power bill. Let's consider a fan that originally used 3.0 BHP that is slowed to half the original speed. Original power usage is 2,238 W of power:

$$(3.0 \text{ BHP})(746 \text{ W/BHP}) = 2,238 \text{ W}$$

The monthly cost, assuming 30 days operation @ 12 hour shifts is

$$2,238 \text{ W} * 360 \text{ h} = 805,680 \text{ W-h} \sim 806 \text{ KWh}$$

At .10 /KWh, this fan's cost is over \$80/mo:



$0.10 / \text{KWh} * 806 \text{ KWh} = \80.60

By slowing the fan down to $\frac{1}{2}$, the cost is $\frac{1}{8}$
or $\$80.60/8 = \10.08 per month.

The savings is therefore \$70.52 per month.¹

Now here's an added bonus to the savings:
Make-Up Air.


The make-up air fan can also be reduced, or
should I say, it MUST be reduced to maintain
proper space pressure.²

Assuming the design included 100% make-up
air (also referred to as *Replacement Air*), it is
common for the total pressure required for the
make-up air fan to be around 0.5 – 0.80 in
WC, depending on what equipment is being
used (evaporative cooling, direct expansion
refrigeration (DX), chilled water air handler,
etc.) Let's assume that the make-up air fan
comes to $\frac{1}{2}$ of the exhaust fan pressure.
Recalling that fan pressure is proportional to
the square of fan speed or flowrate, the power
used is $\frac{1}{4}$ that of the exhaust fan. We can
further save some calculations by including
make-up air savings in the exhaust fan savings
using this $\frac{1}{4}$ rule. So, the savings should be
multiplied by 1.25 to include the make-up air
fan. Again, we have made some assumptions
that must be considered for each project before
an accurate cost savings can be calculated.

$\text{Savings} = 1.25 * \$70.52 = \$88.15$ per month.

Now that's a nice savings.

So, what is the cost of implementing a variable
speed system? The question was posed to one
of the leading manufacturer's and the answer?
"Around \$1,500". At a savings of \$88.15 per
month, this comes to about 17 months.

In our next issue we discuss additional aspects
of Variable Speed Kitchen Exhaust Fans and
Make-Up Fans. 

¹Author's Note: We are not including all of the
details needed to be exact, which include
motor & drive efficiencies, motor loading
factors, miscellaneous demand charges by the
power purveyor, etc. but the foregoing will
provide simplified theoretical savings. For a
more in-depth analysis, the reader is referred
to the *The Energy Management Handbook* by
Wayne Turner, The Fairmont Press, 2001.

²The 2006 UMC maintains that air pressure
between the kitchen area (the vicinity in which
the hood is installed) and surrounding areas
must not differ by more than 0.02" WC. Note
that a difference in airflow rate is not
provided, but the code goes right to the issue –
pressure differential – and rightly so. We will
explore more on this issue in later white papers
and eNewsletters.